

**IN THE SPECIFICATION:**

Please change paragraph 0018 to read:

[0018] It is preferred and favorable with respect to turbine performance, if the turbine blades are tilted forward out of exact radial alignment of their connecting line between the inner and outer edge, so that the radially outer edge of the turbine blade is somewhat ahead of the inner edge in the direction of rotation. It is also preferred that the angle between the connecting line of the outer and inner edge of a turbine blade be inclined by an angle of between  $5^{\circ}$  and  $12^{\circ}$ , preferably by approximately  $8^{\circ}$  to  $-1^{\circ}$ , relative to the turbine-wheel radius vector directed toward the inner edge of the turbine blade.

Please change paragraph 0022 to read:

[0022] Fig. 1 shows a turbine wheel with carrier plate 1, in the form of a circular disk, which has an external diameter "D." On the outer margin of carrier plate 1 is a ring of turbine blades 2, the outer edges of the turbine blades being located approximately on the outer circumference, whereas the inner edges of the blade wheel ring define an inner diameter "d," as also indicated in Fig. 2. Specifically, the values for "d" are between about 40 and 48 mm, preferably about 44 mm, and the values for "D" are between about 50 and 60 mm, preferably 27.5 mm, the difference "D-d" being approximately 20% of "D." The corresponding radii, of course, are half the respective diameter values.

Please change paragraph 0023 to read:

[0023] As also shown in Fig. 1, the individual blades are tilted forward by an angle " $\alpha$ " relative to the radial direction, such that the outer edge of a turbine blade is somewhat ahead of the inner edge in the direction of rotation "R." Specifically, the angle " $\alpha$ ," which is the angle of

a connecting line of the outer edge and inner edge of a blade relative to the radius vector to the inner edge of the blade, is approximately  $8^{\circ} \pm 1^{\circ}$ . However, the angle “ $\alpha$ ” can be in the range of  $2^{\circ}$  to  $15^{\circ}$ , preferably in the range of  $5^{\circ}$  to  $12^{\circ}$ .

Please change paragraph 0026 to read:

[0026] The transition from the front face to the back face at the inner edge and at the outer edge of turbine blade 2 is effected on the outside via radius of curvature  $R_5$  of approximately 0.15 mm or even somewhat less, and in the region of the inner edge, via radius of curvature  $R_6$ , which is substantially less than 0.1 mm, for example, approximately 0.025 mm, but greater than 0.01 mm.

Please change paragraph 0027 to read:

[0027] Radially inward and outer portions 3b, 3a, respectively, of front face 3 and also corresponding radially inward and outer portions 4b, 4a of back face 4, each have different radii of curvature and also differently disposed centers of curvature. Radially inward portion 4b has radius of curvature  $R_1$  and center of curvature 11. In the case of turbine blade 2 mounted on the turbine wheel, this center of curvature 11 is located somewhat closer to axis 5 of the turbine wheel than is center of curvature 13 of radially inward portion 3b of front face 3. Radially outer portion 3a has a radius of curvature  $R_4$  and inner portion 3b has a radius of curvature  $R_3$ . Center of curvature ~~12~~ 14 in this case is located radially further outside, relative to turbine axis 5, than center of curvature 12 of radially outer portion 4a of the back face. Radially outer portion 4a has a radius of curvature  $R_2$ .

Please change paragraph 0029 to read:

[0029] Overall, for the four radii of curvature of the front and back faces in the preferred version of the invention, the relationship:  $R_4 < R_2 < R_3 < R_1$  applies. The factor between  $R_2$  and

$R_4$  is approximately 1.3, the factor between  $R_3$  and  $R_2$  is approximately 2, and the factor between  $R_1$  and  $R_3$  is approximately 1.1. However, these factors may also vary by 10% in both directions. From the relationships above, and from Fig. 3, it can be deduced that the radii of curvature  $R_1, R_3$  of the inward portions 3b, 4b of the blade are at least 50%, and preferably about 100%, greater than the radii of curvature  $R_2, R_4$  of the radially outer portions 3a, 4a of front faces 3 and back faces 4, respectively. Further, the radius of curvature of radially inward portion 3b, 4b is no more than about four times the radius of curvature of the corresponding radially outer portion 3a, 4a. The radius of curvature of the radially outer portion of the back face 4a is between about 5% and 50% greater than the radius of curvature of the radially outer portion of the front face 3a.

Please change paragraph 0031 to read:

[0031] As can also shown in Figs. 1 and 2, which are represented in the same scale, the length of turbine blade 2 in relation to its radial extent, that is, in relation to the difference between the external radius and the internal radius of the blade ring, is approximately 68%. The preferred range is between about 65% and 70%. If possible, a value of approximately 80%, but at most, approximately 100%, should not be exceeded for the axial length of the blades relative to their radial extent. Stated another way, the axial length of the turbine blades is approximately  $70\% \pm 5\%$  of the radial extent of the turbine blades. In addition to the special profile and the special arrangement of the turbine blades on the turbine wheel, this greater axial length in relation to the radial extent of the blades is also conducive to improved and stable turbine performance, and to a greater torque at high rotational speeds.